replaces: 11.94



Variable Displacement Pump A4VTG

for drum drives on mobile concrete mixers for closed circuits

Sizes 56...90 Series 3 Nominal pressure 400 bar Peak pressure 450 bar



A4VTG...HW

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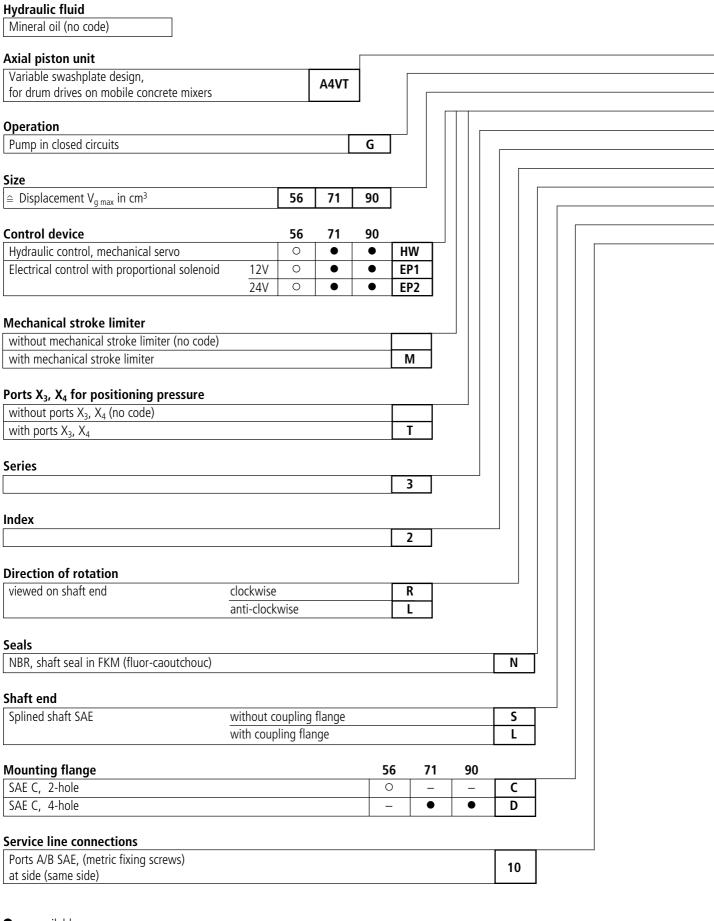
2...3

- variable displacement axial piston pump of swashplate design for hydrostatic closed circuit transmissions
- flow is proportional to drive speed and displacement and is infinitely variable
- 5 output flow increases with swivel angle from 0 to its maximum value
 - swivelling the pump over centre smoothly changes the direction of flow
 - the pump is equipped with two pressure relief valves on the high pressure ports to protect the hydrostatic transmission (pump and motor) from overloads
- 9 these valves also function as boost inlet valves
- 10 an integral auxiliary pump serves as boost and pilot oil pump
 - the maximum boost pressure is limited by a built-in boost pressure relief valve

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A4VTG **1**/12

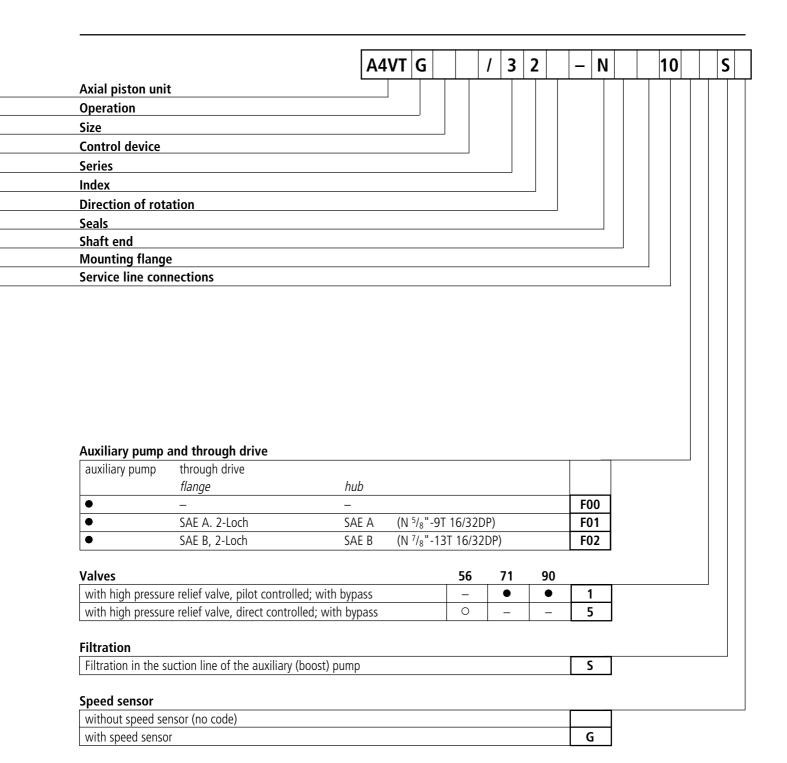
Ordering Code



 \bullet = available

 \bigcirc = in preparation

– not available



Technical Data

Fluid

To review the application of A4VTG pumps with the selected hydraulic fluid, detailed fluid compatibility and application data can be found in data sheets RE 90220 (mineral oil), RE 90221 (environmentally acceptable hydraulic fluids) and RE 90223 (fire resistant fluids, HF).

The variable pump A4VTG is not suitable for operation with HFA, HFB and HFC. When using HFD or environmentally acceptable hydraulic fluids limitations for the technical data have to be taken into consideration. If necessary please consult our technical department (please indicate type of the hydraulic fluid used for your application on the order sheet).

Operating viscosity range

In order to obtain optimum efficiency and service life, we recommend that the operating viscosity (at operating temperature) be selected from within the range:

 v_{opt} = operating viscosity 16...36 mm²/s

referred to the circuit temperature (closed circuit).

Viscosity limits

The limiting values for viscosity are as follows:

 $v_{min} = 5 \text{ mm}^2/\text{s}$

short term at a max. permissible temp. of $t_{max} = 115$ °C.

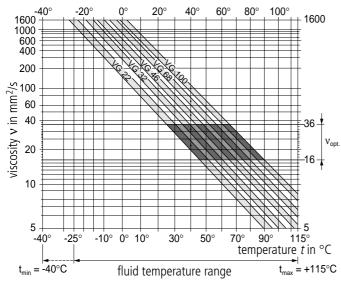
Please note that the max. fluid temperature is also not exceeded in certain areas (for instance bearing area).

 $v_{\text{max}} = 1600 \text{ mm}^2/\text{s}$

short term on cold start ((n \leq 1000 rpm, $t_{min} = -40$ °C).

At temperatures of -25°C up to -40°C special measures are required. Please contact us for further information.

Selection diagram



Notes on the selection of the hydraulic fluid

In order to select the correct fluid, it is necessary to know the operating temperature in the circuit (closed circuit) in relation to the ambient temperature.

The hydraulic fluid should be selected so that within the operating temperature range, the operating viscosity lies within the optimum range (ν_{opt}) (see shaded section of the selection diagram). We recommend that the highest possible viscosity range should be chosen in each case.

Example: At an ambient temperature of X°C circuit temperature is 60°C. Within the operating viscosity range (ν_{opt} ; shaded area) this corresponds to viscosity ranges VG 46 or VG 68. VG 68 should be selected

Important: The leakage oil (case drain oil) temperature is influenced by pressure and pump speed and is always higher than the circuit temperature. However, at no point in the circuit may the temperature exceed 115°C.

If it is not possible to comply with the above conditions because of extreme operating parameters or high ambient temperatures please consult us.

Temperature range of the radial shaft seal

The FKM shaft seal is admissible for a housing temperature range from -25°C to +115°C.

Note:

For applications below -25°C a NBR shaft seal is necessary (admissible temperature range -40°C to +90°C).

When ordering, please state in clear text: with NBR shaft seal

Working pressure range - inlet

Auxiliary pump: suction pressure $p_{s min}$ ($v \le 30 \text{ mm}^2/s$) $\geq 0.8 \text{ bar absolute}$ for cold start $\geq 0.5 \text{ bar absolute}$

Working pressure range - outlet

Case drain pressure

Permissible case drain pressure at ports T_1 and T_2 p_L ______4 bar abs. short term (at start) ______6 bar abs.

Technical Data

Table of values (theoretical values, without considering η_{mh} and η_{v} : values rounded)

Size					71	90
Displacement	variable pump		$V_{g\ max}$	cm ³	71	90
	auxiliary pump	(at $p = 20$ bar)	V_{gH}	cm ³	19,6	28,3
Speed	max. speed wi	th V _{g max}	n _{max contin.}	rpm	3300	3050
	minimum spee	d	n _{min}	rpm	500	500
Flow	at n _{max contin.} ar	ıd V _{g max}	q _{v max}	L/min	234	275
Power	at n _{max contin.}	$\Delta p = 400 \text{ bar}$	P _{max}	kW	156	183
Torque	at V _{g max}	$\Delta p = 400 \text{ bar}$	T _{max}	Nm	451	572
(variable pump withou	t aux. pump)	$\Delta p = 100 \text{ bar}$	T	Nm	112,8	143
Moment of inertia (about drive axis)		J	kgm²	0,0072	0,0106	
Weight (standard model without through drive) approx.		т	kg	46	48	

Pressure Relief Valve

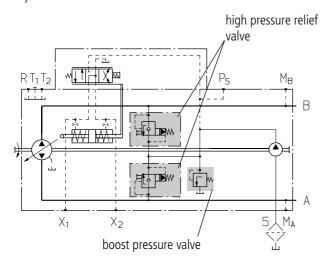
The two high pressure relief valves are protecting the hydrostatic transmission (pump and motor) against overcharge. They are limiting the max. pressure of the respective high pressure line and are serving at the same time for boost valves.

The boost pressure is set via boost pressure valve.

Standard setups:

- boost pressure valve:	P _{SP}	22 bar
- high pressure valve:	p _{max}	420 bar

Note: The valve setups are carried out at $n=1000\ \text{rpm}$ and $V_{g\ \text{max}}$. At other operating parameters differences in the opening pressures may occur.



Filtration

The finer the filtration the better the achieved cleanliness level of the pressure fluid and the longer the life of the axial piston unit.

To ensure the functioning of the axial piston unit a minimum cleanliness level of

9 to NAS 1638

18/15 to ISO/DIS 4406 is necessary.

At very high temperatures of the hydraulic fluid (90°C to max. 115°C) at least cleanliness level

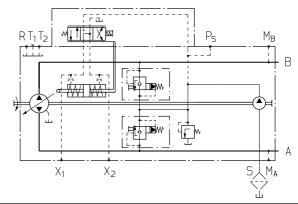
8 to NAS 1638

17/14 to ISO/DIS 4406 necessary.

If above mentioned grades cannot be maintained please consult supplier.

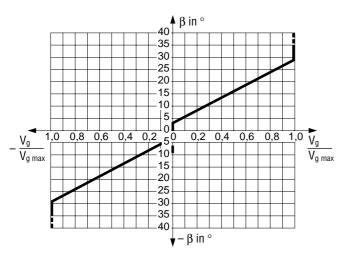
Standard: Filtration in the suction line of the auxiliary pump, S

Filter type: ______ filter **without** bypass Recommendation: _____ **with** contamination indicator Through flow resistance at filter element: at $\nu = 30 \text{ mm}^2/\text{s}$, $n = n_{\text{max}}$ _____ $\Delta p \leq 0,1$ bar at $\nu = 1000 \text{ mm}^2/\text{s}$, $n = n_{\text{max}}$ _____ $\Delta p \leq 0,3$ bar Pressure at port S of the auxiliary pump: at $\nu = 30 \text{ mm}^2/\text{s}$ ______ $p \geq 0,8$ bar at cold start ($\nu = 1600 \text{ mm}^2/\text{s}$, $n \leq 1000 \text{ rpm}$) ______ $p \geq 0,5$ bar



HW Hydraulic Control, Mechanical Servo

The positioning cylinder of the pump and therefore the swivel angle is varied in proportion to the movement of the control lever. The pump control is steplessly variable. Each direction of flow is assigned to one direction of lever movement.



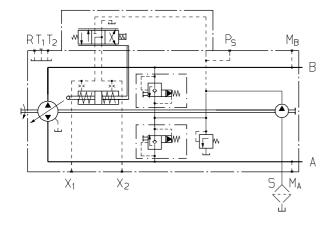
Swivel angle of control lever:

- from 0 to \pm V $_{g\;max}$	$\beta = 0^{\circ} \text{ to } \pm 29^{\circ}$
- mech. stop:	± 40°

Torque at control lever

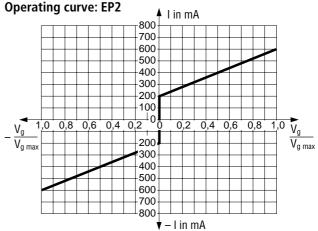
- necessary torque for displacement _____approx. 85 to 210 Ncm

Dependant from the operation conditions of the pump (operation pressure, oil temperature) changes of the curve can occur.



EP Electrical Control, with Proportional Solenoid

In relation to the preselected current, control pressure is applied to the positioning cylinder of the pump via two proportional solenoids on control device EP. The displacement of the pump is thus steplessly variable. One solenoid is assigned to each direction of flow.

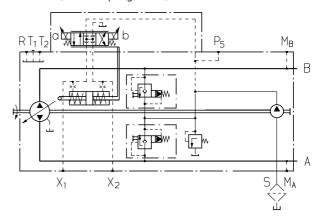


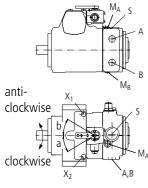
model	control voltage (DC)	control current / start of control — end of con	
	,	at V_{g0}	at $V_{g max}$
EP1	12 V	400 mA —	1200 mA
EP2	24 V	200 mA —	600 mA

To control the proportional solenoids the following electronic amplifiers and microcontroller are available:

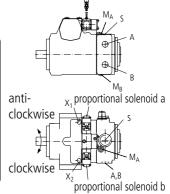
proportional amplifier PVR (RE 95022), chopper amplifier CV (RE 95029) and microcontroller MC with software solutions related to the field of application (RE 95050) (e.g. CSD).

Standard: proportional solenoid with manual emergency (without spring return)



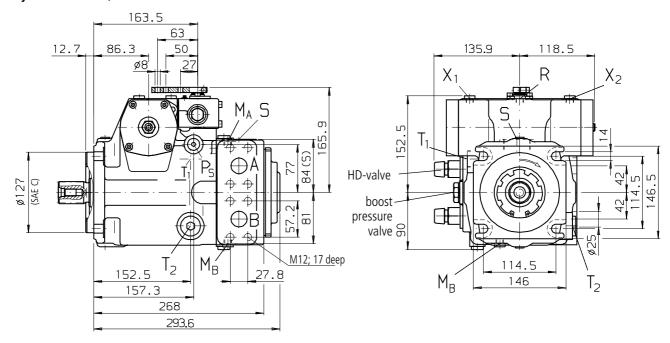


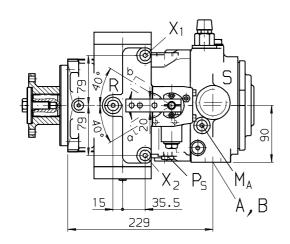
Graph Direction of rotation - Control - Direction of through flow						
Direction of rotation	clock	wise	anti-clockwise			
Lever direction (HW)	а	b	а	b		
Operated solenoid (EP)	b	а	b	a		
Control pressure	X ₂	X ₁	X ₂	X ₁		
Direction flow	B to A	A to B	A to B	B to A		
Working pressure	M _A	M _B	M _B	M _A		



Prior to finalising your design, please obtain a certified drawing.

Hydraulic control, mechanical servo HW





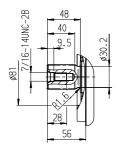
Connections

A,B	Service line ports SAE 1"
	420 bar (6000 psi) (high pressure series)

T_1	Case drain or filling port	M 26x1,5; 16 deep
T_2	Case drain	M 26x1,5; 16 deep
M_A	Pressure gauge - working pressure A	M 12x1,5; 12 deep
M_{B}	Pressure gauge - working pressure B	M 12x1,5; 12 deep
R	Air bleed	M 16x1,5; 12 deep
S	Boost suction port	M 42x2; 18 deep
X ₁ , X ₂	Control pressure ports (before the orifice)	M 12x1,5; 12 deep
p_{S}	Control pressure supply	M 14x1,5; 12 deep

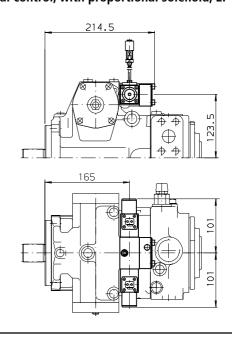
Shaft ends

detail A-A L Splined shaft SAE 1³/₈" with coupling flange



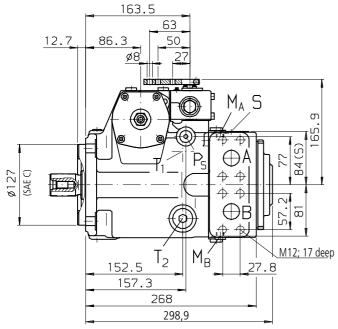
Splined shaft SAE 1³/₈" pressure angle 30°, 21 teeth, 16/32 pitch, flat root side fit, tolerance class 5 ANSI B92.1a-1976

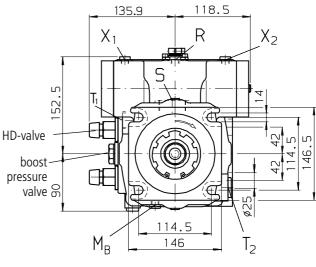
Electrical control, with proportional solenoid, EP



Hydraulic control, mechanical servo HW

Prior to finalising your design, please obtain a certified drawing.





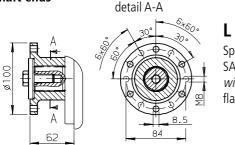
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Connections

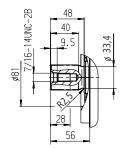
A,B Service line ports SAE 1" 420 bar (6000 psi) (high pressure series)

T_1	Case drain or filling port	M 26x1,5;	16 deep
T_2	Case drain	M 26x1,5;	16 deep
M_A	Pressure gauge - working pressure A	M 12x1,5;	12 deep
M_{B}	Pressure gauge - working pressure B	M 12x1,5;	12 deep
R	Air bleed	M 16x1,5;	12 deep
S	Boost suction port	M 42x2;	18 deep
X ₁ , X ₂	Control pressure ports (before the orifice)	M 12x1,5;	12 deep
Dc	Control pressure supply	M 14x1.5:	12 deep



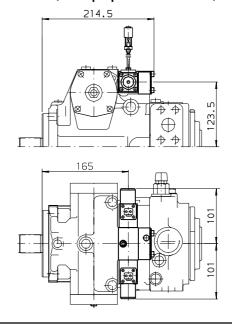


Splined shaft SAE 1¹/₂" with coupling flange



S Splined shaft SAE 1¹/₂" pressure angle 30°, 23 teeth, 16/32 pitch, flat root side fit, tolerance class 5 ANSI B92.1a-1976

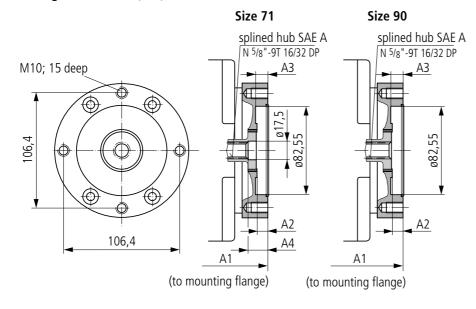
Electrical control, with proportional solenoid, EP



Dimensions for Through Drives

Through drive SAE A (F01)

Prior to finalising your design, please obtain a certified drawing.

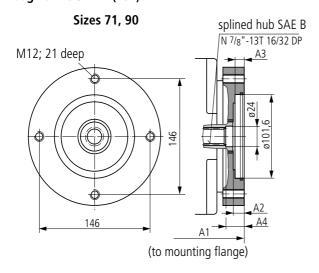


Size	A1	A2	A3	A4
71	297,6	9	10	17
90	297,6	9	8	_

suitable for connection of:

- gear pump G2 (RE 10030)
- variable pump A10VSO10 (RE 92713)
- variable pump A10VSO18 (RE 92712)

Through drive SAE B (F02)



Size	A1	A2	А3	A4
71	300,6	12,5	9,8	15,5
90	313,6	13	9,8	19

suitable for connection of:

- gear pump G3 (RE 10039)
- gear pump G4 (RE 10042)
- variable pump A10VG18 (RE 92750)
- variable pump A10VO28 (RE 92701/ RE 92703)

Permissible Input and Through Drive Rotation Torques

Size			71	90
Corner torque (when V_{gmax} and $\Delta p = 400$ bar) 1)	T _{max}	Nm	451	572
Max. perm. through drive rotation torque	T _{D perm.}	Nm	660	822
Max. permissible input torque ²)	T _{E perm.}	Nm	970	1305
	·		(W 1 3/8")	(W 1 ¹ / ₂ ")

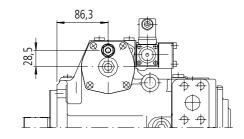
- 1) efficiency not taken into consideration
- 2) drive shaft without side load

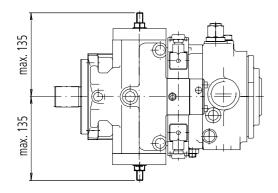
Code explanations

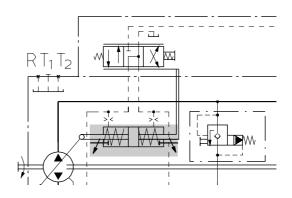
 $T_{D perm.}$ = max. permissible through drive torque in Nm $T_{E perm.}$ = max. permissible input torque at the drive shaft in Nm

Mechanical Stroke Limiter, M

Adjustment screws to both $\rm V_{g\;max} - values$

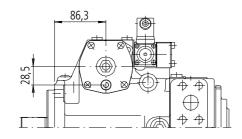


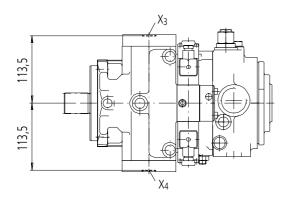


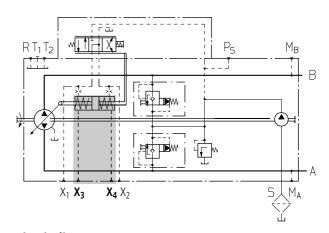


Circuit diagram A4VTG 71HWM

Ports $\mathbf{X_3}$ and $\mathbf{X_4}$ for Positioning Pressure, T







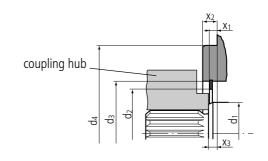
Circuit diagram A4VTG 71HWT

Installation Situation for Coupling Assembly

In order to assure that rotating parts (coupling hub) and fixed parts (housing, circlip) do not contact each other the installation situations are described in this leaflet have to be observed.

At design "shaft end L" (SAE splined shaft with mounted coupling flange) Brueninghaus Hydromatik already considered this fact.

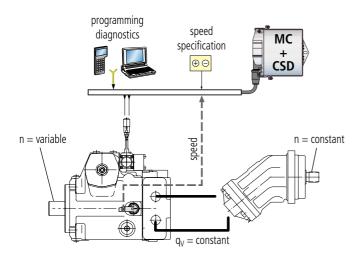
At design "shaft end S" (SAE splined shaft) it has to be considered when selecting the coupling, that the external diameter of the coupling hub in the area of the shaft collar (size $x_2 - x_3$) has to be smaller than the internal diameter of the circlip d_2 .



Size	ϕd_1	$\mathrm{ød}_{\mathrm{2min}}$	ϕd_3	$ød_4$	$\mathbf{x_1}$	\mathbf{x}_{2}	X_3
71	45	66,5 66,5	81 _{±0,1}	127	7,0+0,2	12,7 _{-0,5}	8 +0,9
90	45	66,5	81 _{±0,1}	127	7,0+0,2	12,7 _{-0,5}	O _{-0,6}

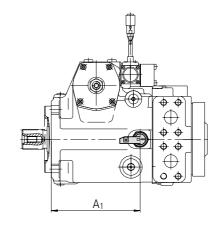
Speed Sensor, G

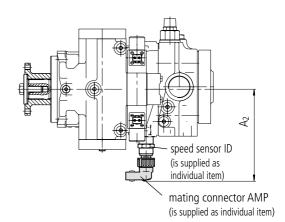
The design "with speed sensor G" is facilitating at electronically piloted drives a speed record of the variable pump A4VTG. In connection with a microcontroller MC (RE 95050) and software CSD an alternating input speed of the pump can thus be picked up and the constant flow $q_{\rm V}$ is resulting. Consequently a constant output speed (mixer drum speed) is generated at the output shaft of the so driven hydromotor.



The design "with speed sensor G" is including a gears spline at the rotary group of the variable pump and an additional port for the speed record, in which an inductive speed sensor type IDR 18/20-L250 (as to RE 95130) is screwed in.

Note: The sensor inclusively necessary mating connector (mating connector for IDR elbow) is supplied as individual item and has to be screwed into the provided port before start-up. To avoid measurment errors, the max. tightening torque of 50 Nm should not be exceeded.





Size	71	90	
No. of teath	46	50	
A ₁	176	176	
A ₂	170	170	

Installation and Commissioning Guidelines

General

At start-up and during operation the pump housing has imperatively to be filled up with hydraulic fluid (filling of the case chamber). Start-up has to be carried out at low speed and without load till the system is completely bleeded.

At a longer standstill the case may discharge via operating line. At new start-up a sufficient filling of the housing has to be granted.

The leakage oil in the housing has to be discharged to the tank via highest positioned case drain port. The min. suction pressure at port S should not fall below 0,8 bar absolute (cold start 0,5 bar absolute).

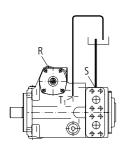
Installation position

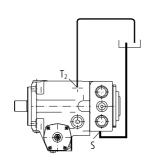
Shaft horizontal, further installation positions as to agreement.

Installation below tank level

Pumps below min. oil level in the tank (standard)

- → Fill up axial piston pump before start-up via highest positioned case drain port
- → Recommendation: Fill up suction lines.
- → Operate pump at low speed (igniton speed) till pump system is completely filled up (Verification: oil is leaking out without bubbles at port p_s, discharge oil via gauge pipe into reservoir).
- → Bleed pump at installation position "control device on top" via port R.
- → Bleed closed loop:
 - variable motor A6VM: via port G
 - fixed motor A2FM: via operating port A, B
 - motor with flushing valve: bleeding not necessary
- → Minimal depth of immersion of the suction resp. case drain line in the tank: 200 mm (corresponding to the min. oil level in the tank)

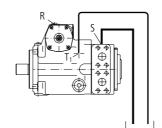


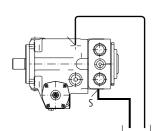


Installation on top of tank level

Pump on top of min. oil level in the tank

- → Actions as installation below tank level
- → Note: max. admissible suction height $h_{max} = 800 \text{ mm}$
 - min. admissible presssure at port S (min. suction pressure)





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